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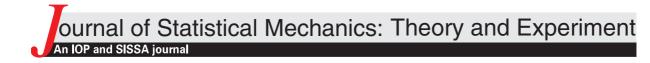
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Received 16 September 2010 Accepted 3 November 2010 Published 1 December 2010

Online at stacks.iop.org/JSTAT/2010/P12003 doi:10.1088/1742-5468/2010/12/P12003

**Abstract.** We introduce from an experimental point of view the main concepts of fluctuation theorems for work, heat and entropy production in out-of-equilibrium systems. We will discuss the important difference between the applications of these concepts to stochastic systems and to a second class of systems (chaotic systems) where the fluctuations are induced either by chaotic flows or by fluctuating driving forces. We will mainly analyze the stochastic systems using the measurements performed in two experiments: (a) a harmonic oscillator driven out of equilibrium by an external force; (b) a colloidal particle trapped in a time-dependent double-well potential. We will rapidly describe some consequences of fluctuation theorems and some useful applications to the analysis of experimental data. As an example, the case of a molecular motor will be analyzed in some detail. Finally we will discuss the problems related to the applications of fluctuation theorems to chaotic systems.

**Keywords:** stochastic particle dynamics (experiment), Brownian motion, current fluctuations, large deviations in non-equilibrium systems

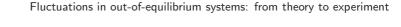
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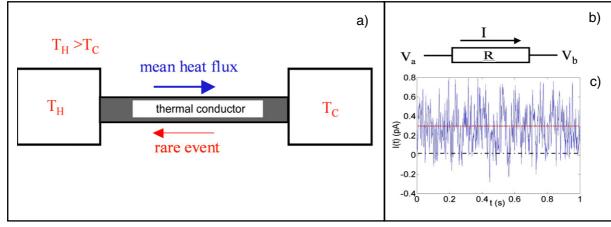
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## 1. Introduction

This article is a review of the main experimental applications of fluctuation theorems (FTs) and summarizes the plenary talk given at STATPHYS24. In order to define the main contents let us consider several simple examples. The simplest and most basic outof-equilibrium system is a thermal conductor whose extremities are connected to two heat baths at different temperatures, as sketched in figure 1. The second law of thermodynamics imposes that on average the heat flows from the hot to the cold reservoir (from H to C in figure 1). However the second law does not say anything about fluctuations and in principle one can observe for a short time a heat current in the opposite direction. What is the probability of observing these rare events? As a general rule when the size of the system decreases the role of fluctuations increases. Thus from an experimental point of view it is reasonable to think that such rare events can be observed in systems that are

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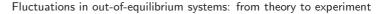


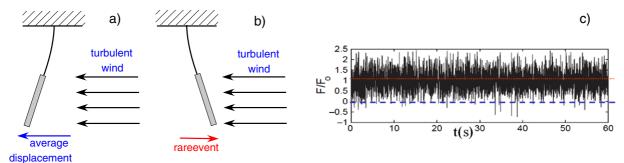


**Figure 1.** (a) Schematic representation of a conductor whose extremities are in contact with two heat baths at temperatures  $T_{\rm H}$  and  $T_{\rm C}$  with  $T_{\rm H} > T_{\rm C}$ . (b) Electrical analogy. A conductor of electrical resistance R and kept at a temperature T is subjected to a potential difference  $V = V_a - V_b$ . (c) Instantaneous current I flowing into the resistance using  $R = 10 \text{ M}\Omega$ , T = 300 K and  $\tau_0 = 2 \text{ ms}$ .

small. A good candidate could be for example the thermal conduction in a nanotube whose extremities are connected to two heat baths [1], exactly in the spirit of figure 1. In reality in these kinds of experiments the measuring of the mean quantities [1] is already difficult and of course the analysis of fluctuations is even more complicated. However there is an electrical analogy, shown in figure 1(b), of the thermal model of figure 1(a). Let us consider an electrical conductor connected to a potential difference  $V = V_A - V_B$ and kept at temperature T by a heat bath. If the mean current  $\bar{I} = V/R$  (R being the electrical resistance of the conductor) is of the order of  $10^{-13}$  A and the injected power is about  $100k_{\rm B}T \simeq 10^{-19}$  J ( $k_{\rm B}$  is the Boltzmann constant) then the instantaneous current inside the resistance has fluctuations whose amplitude is comparable to the mean, as shown in figure 1(c). The variance of these fluctuations is  $\delta I^2 \simeq k_{\rm B}T/(R\tau_0)$  where  $\tau_0$  is the characteristic time constant of the electrical circuit. In the specific case of figure 1(c) the current reverses with respect to the mean value. The probabilities of having those negative currents have been studied both theoretically and experimentally in [2,3] within the context of fluctuation theorems, that we will present in section 3.

We discuss a second example where the source of fluctuation is not the coupling with a thermal bath, as in the case of the electrical conductor, but is either a chaotic flow or a chaotic force produced by the non-linear interaction of many degrees of freedom of a dissipative system sustained by an external driving. We will refer to them as chaotic systems. Let us consider a turbulent wind flowing around an object as sketched in figures 2(a) and (b). The wind exerts a mean force  $F_0$  on the object but the instantaneous force, plotted in figure 2(c), is a strongly fluctuating quantity which presents negative values [4], i.e. the object moves against the wind; figure 2(b). In such a case the mean work done by the wind on the object is about 0.1 J  $\simeq 10^{20} k_{\rm B}T$ , and obviously thermal fluctuations do not play any role, but the chaotic flow does, which produces





**Figure 2.** (a), (b) Schematic representation of an object suspended by an elastic beam and subjected to the pressure of a turbulent wind: (a) average behavior; (b) rare event. (c) Time evolution of the measured instantaneous force exerted by the turbulent wind on the object. The details of this experiment can be found in [4].

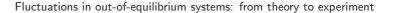
the fluctuations. Other similar examples can be found in for example shaken granular media [5, 6], discussed in section 6.

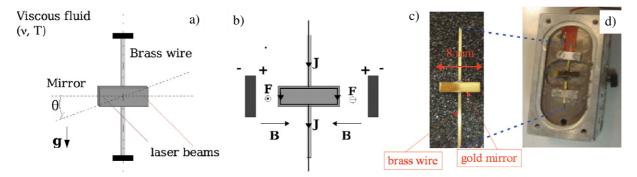
These examples stress that in the two experiments on the electrical conductor and the turbulent wind we may observe the counterintuitive effect that the instantaneous response of the system is opposite to the mean value, in other words the system has an instantaneous negative entropy production rate. This effect is induced by the thermal fluctuations in the first case and by the chaotic flow in the second case. The question that we want analyze in this paper is that of whether the fluctuation theorem (defined in section 3) is able to predict the probability of these rare events in both cases, i.e. for the stochastic and the chaotic systems. We will take an experimentalist approach and we will use experimental results in order to introduce the main concepts.

The largest part of the paper concerns stochastic systems described by a Langevin dynamics. For chaotic systems we will mainly discuss the difficulty of comparing the experimental results with the theoretical predictions. The paper is organized as follows. In section 2 we present the experimental results on the energy fluctuations measured in a harmonic oscillator driven out of equilibrium by an external force. In section 3 the experimental results on the harmonic oscillator are used to introduce the properties of fluctuation theorems (FTs). In section 4 the non-linear case of a Brownian particle confined in a time-dependent double-well potential is presented. In section 5 we introduce the applications of the FT, and as a more specific example we describe the measuring of the torque of a molecular motor. Finally in section 6 we discuss the chaotic systems and we conclude in section 7.

#### 2. Work and heat fluctuations in the harmonic oscillator

The choice of discussing the dynamics of the harmonic oscillator is dictated by the fact that it is relevant for many practical applications such as the measuring of the elasticity of nanotubes [7], the dynamics of the tip of an AFM [8], the MEMS and the thermal rheometer that we developed several years ago to study the rheology of complex fluids [9, 10].





**Figure 3.** (a) The torsion pendulum. (b) The magnetostatic forcing. (c) A picture of the pendulum. (d) The cell where the pendulum is installed.

#### 2.1. The experimental set-up

This device is a very sensitive torsion pendulum as sketched in figure 3(a). It is composed of a brass wire (length 10 mm, width 0.5 mm, thickness 50  $\mu$ m) and a glass mirror with a golden surface, figure 3(c). The mirror (length 2 mm, width 8 mm, thickness 1 mm) is glued to the middle of the brass wire. The elastic torsional stiffness of the wire is  $C = 4.65 \times 10^{-4}$  N m rad<sup>-1</sup>. It is enclosed in a cell (figure 3(d)) which is filled by a fluid. We used either air or a water–glycerol mixture at 60% concentration. The system is a harmonic oscillator with resonant frequency  $f_o = \sqrt{C/I_{\text{eff}}}/(2\pi) = \omega_0/(2\pi)$  and a relaxation time  $\tau_{\alpha} = 2I_{\text{eff}}/\nu = 1/\alpha$ .  $I_{\text{eff}}$  is the total moment of inertia of the displaced masses (i.e. the mirror and the mass of displaced fluid) [11]. The damping has two contributions: the viscous damping  $\nu$  of the surrounding fluid and the viscoelasticity of the brass wire.

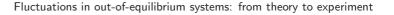
The angular displacement of the pendulum  $\theta$  is measured by using a differential interferometer [12]–[15] which uses the two laser beams reflected by the mirror figure 3(a). The measurement noise is two orders of magnitude smaller than the thermal fluctuations of the pendulum.  $\theta(t)$  is acquired with a resolution of 24 bits at a sampling rate of 8192 Hz, which is about 40 times  $f_0$ . We drive the system out of equilibrium by forcing it with an external torque M by means of a small electric current J flowing in a coil glued behind the mirror (figure 3(b)). The coil is inside a static magnetic field. The displacements of the coil and therefore the angular displacements of the mirror are much smaller than the spatial scale of inhomogeneity of the magnetic field. So the torque is proportional to the injected current: M = AJ; the slope A depends on the geometry of the system. The practical realization of the montage is shown in figures 3(c) and (d). In equilibrium the variance  $\delta\theta^2$  of the thermal fluctuations of  $\theta$  can be obtained from equipartition, i.e.  $\delta\theta = \sqrt{k_{\rm B}T/C} \simeq 2$  nrad for our pendulum, where T is the temperature of the surrounding fluid.

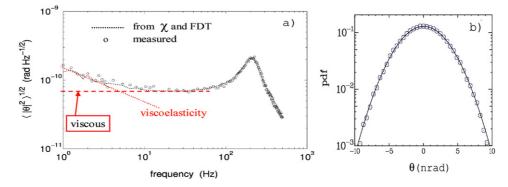
#### 2.2. The equation of motion

The dynamics of the torsion pendulum can be likened to that of a harmonic oscillator damped by the viscoelasticity of the torsion wire and the viscosity of the surrounding fluid, whose equation of motion reads in the temporal domain

$$I_{\text{eff}} \ddot{\theta} + \int_{-\infty}^{t} G(t - t') \dot{\theta}(t') \,\mathrm{d}t' + C\theta = M + \eta, \tag{1}$$

doi:10.1088/1742-5468/2010/12/P12003





**Figure 4.** Equilibrium: the pendulum inside a glycerol-water mixture with M = 0. (a) Square root of the power spectral density of  $\theta$ .  $\bigcirc$ : directly measured spectrum; the black dotted line is the spectrum estimated from the measurement of  $\chi$  and using equation (3). The red dashed and dotted lines show the viscous and viscoelastic components of the damping respectively. (b) Probability density function of  $\theta$ . The continuous line is a Gaussian fit.

where G is the memory kernel and  $\eta$  the thermal noise. In Fourier space (in the frequency range of our interest) this equation takes the simple form

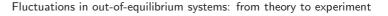
$$\left[-I_{\text{eff}}\,\omega^2 + \hat{C}\right]\hat{\theta} = \hat{M} + \eta,\tag{2}$$

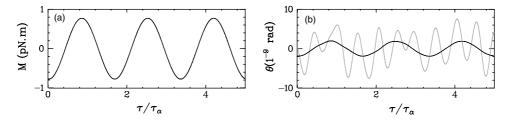
where  $\hat{\cdot}$  denotes the Fourier transform and  $\hat{C} = C + i[C_1'' + \omega C_2'']$  is the complex frequencydependent elastic stiffness of the system.  $C_1''$  and  $C_2''$  are the viscoelastic and viscous components of the damping term.

2.2.1. Equilibrium. At equilibrium, i.e. M = 0, the fluctuation-dissipation theorem (FDT) gives a relation between the amplitude of the thermal angular fluctuations of the oscillator and its response function. The response function of the system  $\hat{\chi} = \hat{\theta}/\hat{M} = (\hat{\theta}/A\hat{J})$  can be measured by applying a torque with a white spectrum. When M = 0, the amplitude of the thermal vibrations of the oscillator is related to its response function via the fluctuation-dissipation theorem (FDT). Therefore, the thermal fluctuation power spectral density (psd) of the torsion pendulum reads for positive frequencies

$$\langle |\hat{\theta}|^2 \rangle = \frac{4k_{\rm B}T}{\omega} \, \mathrm{Im} \, \hat{\chi} = \frac{4k_{\rm B}T}{\omega} \frac{C_1'' + \omega \, C_2''}{\left[ -I_{\rm eff} \, \omega^2 + C \right]^2 + \left[ C_1'' + \omega \, C_2'' \right]^2}.$$
(3)

The brackets are ensemble averages. As an example, the spectrum of  $\theta$  measured in the glycerol-water mixture is shown in figure 4(a). In this case the resonance frequency is  $f_{\rm o} = \sqrt{C/I_{\rm eff}}/(2\pi) = \omega_0/(2\pi) = 217$  Hz and the relaxation time  $\tau_{\alpha} = 2I_{\rm eff}/\nu = 1/\alpha = 9.5$  ms. The measured spectrum is compared with that obtained from equation (3) using the measured  $\chi$ . The viscoelastic component at low frequencies corresponds to a constant  $C_1'' \neq 0$ . Indeed if  $\omega \to 0$ , then from equation (3),  $\langle |\hat{\theta}|^2 \rangle \propto 1/\omega$  as seen in figure 4(a). Instead, if  $C_1'' = 0$ , then for  $\omega \to 0$  from equation (3) the spectrum is constant as a function of  $\omega$ . It is important to stress that in the viscoelastic case the noise  $\eta$  is correlated and the process is not Markovian, whereas in the viscous case the process is Markovian. Thus by changing the quality of the fluid surrounding the pendulum one can tune the





**Figure 5.** (a) Sinusoidal driving torque applied to the oscillator. (b) Response of the oscillator to this periodic forcing (gray line); the dark line represents the mean response  $\langle \theta(t) \rangle$ .

Markovian nature of the process. In the following we will consider only the experiment in the glycerol-water mixture where the viscoelastic contribution is visible only at very low frequencies and is therefore negligible. This allows a more precise comparison with theoretical predictions often obtained for Markovian processes. The probability density function (pdf) of  $\theta$ , plotted in figure 4(b), is a Gaussian.

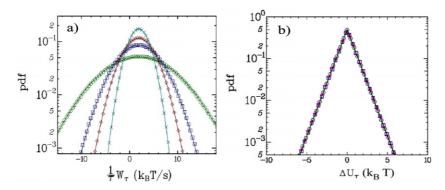
#### 2.3. The non-equilibrium steady state (NESS): sinusoidal forcing

We now consider a periodic forcing of amplitude  $M_{\rm o}$  and frequency  $\omega_{\rm d}$ , i.e.  $M(t) = M_{\rm o} \sin(\omega_{\rm d} t)$  [14]–[17]. This is a very common kind of forcing which has already been studied in the case of the first-order Langevin equation [18] and of the two-level system [19] and in a different context for the second-order Langevin equation [20]. Furthermore this is a very general case because using the Fourier transform, any periodic forcing can be decomposed into a sum of sinusoidal forcings. We explain here the behavior of a single mode. Experiments have been performed at various values of  $M_{\rm o}$  and  $\omega_{\rm d}$ . We present here the results for a particular amplitude and frequency:  $M_{\rm o} = 0.78$  pN m and  $\omega_{\rm d}/(2\pi) = 64$  Hz. This torque is plotted in figure 5(a). The mean of the response to this torque is sinusoidal, with the same frequency, as can be seen in figure 5(b). The system is clearly in a non-equilibrium steady state (NESS).

The work done by the torque M(t) in a time  $\tau_n = 2\pi n/\omega_d$  is

$$W_n = W_{\tau = \tau_n} = \int_{t_i}^{t_i + \tau_n} M(t) \frac{\mathrm{d}\theta}{\mathrm{d}t} \,\mathrm{d}t. \tag{4}$$

As  $\theta$  fluctuates,  $W_n$  is also a fluctuating quantity whose probability density function (pdf) is plotted in figure 6(a) for various values of n. This plot has interesting features. Specifically, work fluctuations are Gaussian for all values of n, and  $W_{\tau}$  takes negative values as long as  $\tau_n$  is not too large. The probability of having negative values of  $W_{\tau}$ decreases when  $\tau_n$  is increased. There is a finite probability of having negative values of the work; in other words the system may have an instantaneous negative entropy production rate although the average of the work  $\langle W_n \rangle$  is of course positive ( $\langle \cdot \rangle$  stands for the ensemble average). In this specific example we have  $\langle W_n \rangle = 0.04n(k_{\rm B}T)$ . We now consider the energy balance for the system.



**Figure 6.** Sinusoidal forcing. (a) The pdf of  $W_{\tau}$ ; n = 7 ( $\bigcirc$ ), n = 15 ( $\square$ ), n = 25 ( $\diamondsuit$ ) and n = 50 ( $\times$ ). (b) The pdf of  $\Delta U_{\tau}$ .

#### 2.4. The energy balance

As the fluid is rather viscous we will take into account only the standard viscosity, that is  $C_1'' = 0$  and  $C_2'' = \nu$ . In such a case, equation (1) simplifies to

$$I_{\text{eff}} \frac{\mathrm{d}^2 \theta}{\mathrm{d}t^2} + \nu \frac{\mathrm{d}\theta}{\mathrm{d}t} + C \,\theta = M + \eta, \tag{5}$$

where  $\eta$  is the thermal noise, which in this case is delta correlated in time:  $\langle \eta(t)\eta(t')\rangle = 2k_{\rm B}T\nu\delta(t-t')$ .

When the system is driven out of equilibrium using a deterministic torque, it receives some work and a fraction of this energy is dissipated into the heat bath. Multiplying equation (5) by  $\dot{\theta}$  and integrating between  $t_i$  and  $t_i + \tau$ , one obtains a formulation of the first law of thermodynamics between the two states at time  $t_i$  and  $t_i + \tau$  (equation (6)). This formulation was first proposed in [21] and used in other theoretical and experimental works [22, 18]. The change in internal energy  $\Delta U_{\tau}$  of the oscillator over a time  $\tau$ , starting at a time  $t_i$ , is written as

$$\Delta U_{\tau} = U(t_i + \tau) - U(t_i) = W_{\tau} - Q_{\tau}$$
(6)

where  $W_{\tau}$  is the work done on the system over a time  $\tau$ :

$$W_{\tau} = \int_{t_i}^{t_i + \tau} M(t') \frac{\mathrm{d}\theta}{\mathrm{d}t}(t') \,\mathrm{d}t' \tag{7}$$

and  $Q_{\tau}$  is the heat dissipated by the system. The internal energy is the sum of the potential energy and the kinetic energy:

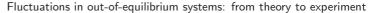
$$U(t) = \frac{1}{2} I_{\text{eff}} \left[ \frac{\mathrm{d}\theta}{\mathrm{d}t}(t) \right]^2 + \frac{1}{2} C \theta(t)^2.$$
(8)

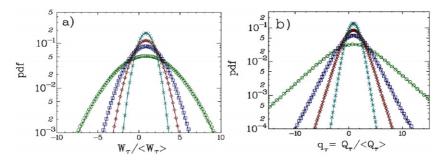
The heat transfer  $Q_{\tau}$  is deduced from equation (6); it has two contributions:

$$Q_{\tau} = W_{\tau} - \Delta U_{\tau}$$
  
=  $\int_{t_i}^{t_i + \tau} \nu \left[ \frac{\mathrm{d}\theta}{\mathrm{d}t}(t') \right]^2 \mathrm{d}t' - \int_{t_i}^{t_i + \tau} \eta(t') \frac{\mathrm{d}\theta}{\mathrm{d}t}(t') \mathrm{d}t'.$  (9)

The first term corresponds to the viscous dissipation and is always positive, whereas the second term can be interpreted as the work of the thermal noise which has a fluctuating sign. The second law of thermodynamics imposes that  $\langle Q_{\tau} \rangle$  is positive.

doi:10.1088/1742-5468/2010/12/P12003





**Figure 7.** Sinusoidal forcing. (a) The pdf of  $W_{\tau}$ . (b) The pdf of  $Q_{\tau}$  for various values of n: n = 7 ( $\bigcirc$ ), n = 15 ( $\square$ ), n = 25 ( $\diamondsuit$ ) and n = 50 ( $\times$ ). The continuous lines in these figures are not fits but analytical predictions obtained from the Langevin dynamics as discussed in section 3.4.

#### 2.5. Heat fluctuations

The dissipated heat  $Q_{\tau}$  cannot be directly measured because we have seen that equation (9) contains the work of the noise (the heat bath) that, experimentally, is impossible to measure, because  $\eta$  is unknown. However  $Q_{\tau}$  can be obtained indirectly from the measuring of  $W_{\tau}$  and  $\Delta U_{\tau}$ , whose pdfs measured during the periodic forcing are exponential for any  $\tau$ , as shown in figure 6(b). We first make some comments on the average values. The average of  $\Delta U_{\tau}$  is obviously vanishing because the time  $\tau$  is a multiple of the period of the forcing. Therefore  $\langle W_n \rangle$  and  $\langle Q_n \rangle$  are equal.

We rescale the work  $W_{\tau}$  (the heat  $Q_{\tau}$ ) by the average work  $\langle W_{\tau} \rangle$  (the average heat  $\langle Q_{\tau} \rangle$ ) and define  $w_{\tau} = (W_{\tau}/\langle W_{\tau} \rangle) (q_{\tau} = (Q_{\tau}/\langle Q_{\tau} \rangle))$ . In the present paper,  $x_{\tau}$  (resp.,  $X_{\tau}$ ) stands for either  $w_{\tau}$  or  $q_{\tau}$  (resp.,  $W_{\tau}$  or  $Q_{\tau}$ ).

We compare now the pdfs of  $w_{\tau}$  and  $q_{\tau}$  in figure 7. The pdfs of heat fluctuations  $q_n$  have exponential tails (figure 7(b)). It is interesting to note that although the two variables  $W_{\tau}$  and  $Q_{\tau}$  have the same mean values they have very different pdfs. The pdfs of  $w_{\tau}$  are Gaussian whereas those of  $q_{\tau}$  are exponential. In a first approximation the pdfs of  $q_{\tau}$  are the convolutions of a Gaussian (the pdf of  $W_{\tau}$ ) and an exponential (the pdf of  $\Delta U_{\tau}$ ). In figures 7 the continuous lines are analytical predictions obtained from the Langevin dynamics with no adjustable parameter (see section 3.4).

#### 3. The fluctuation theorem

In section 2.5 we have seen that both  $W_{\tau}$  and  $Q_{\tau}$  present negative values, i.e. the second law is verified only on average and the entropy production can have instantaneously negative values. The probabilities of getting positive and negative entropy production are quantitatively related in non-equilibrium systems by the fluctuation theorem (FTs).

There are two classes of FTs. The stationary state fluctuation theorem (SSFT) considers a non-equilibrium steady state. The transient fluctuation theorem (TFT) describes transient non-equilibrium states where  $\tau$  measures the time since the system left the equilibrium state. A fluctuation relation (FR) examines the symmetry around zero of the probability density function (pdf)  $p(x_{\tau})$  of a quantity  $x_{\tau}$ , as defined in section 2.5. It compares the probability of having a positive event  $(x_{\tau} = +x)$  versus the probability of

having a negative event  $(x_{\tau} = -x)$ . We quantify the FT using a function S (a symmetry function):

$$S(x_{\tau}) = \frac{k_{\rm B}T}{\langle X_{\tau} \rangle} \ln\left(\frac{p(x_{\tau} = +x)}{p(x_{\tau} = -x)}\right).$$
(10)

The transient fluctuation theorem (TFT) states that the symmetry function is linear with  $x_{\tau}$  for any values of the time integration  $\tau$  and the proportionality coefficient is equal to 1 for any value of  $\tau$ :

$$S(x_{\tau}) = x_{\tau}, \qquad \forall x_{\tau}, \quad \forall \tau. \tag{11}$$

In contrast to the TFT, the stationary state fluctuation theorem (SSFT) holds only in the limit of infinite time  $(\tau)$ :

$$\lim_{\tau \to 0} S(x_{\tau}) = x_{\tau}.$$
(12)

In the following we will assume linearity at finite time  $\tau$  [2, 29] and use the following general expression:

$$S(x_{\tau}) = \Sigma_x(\tau) x_{\tau} \tag{13}$$

where for SSFT  $\Sigma_x(\tau)$  takes into account the finite time corrections and  $\lim_{\tau\to\infty} \Sigma_x(\tau) = 1$ whereas  $\Sigma_x(\tau) = 1, \forall \tau$  for the TFT.

However these claims are not universal because they depend on the kind of  $x_{\tau}$  which is used. Specifically we will see in the following sections that the results are not exactly the same if  $X_{\tau}$  is replaced by any one of  $W_{\tau}$ ,  $Q_{\tau}$  and  $(Ts_{\text{tot},\tau})$ , defined in section 3.3. Furthermore the definitions given in this section are appropriate for stochastic systems and in section 6 we will discuss the differences between stochastic and chaotic systems.

#### 3.1. A short history of FTs

The first numerical evidence of relations of this kind was given by Evans *et al* in [23] whereas the TFT was proved in [24]. In 1995 Gallavotti and Cohen [25] proved the SSFT for dynamical systems although in such a case  $x_{\tau}$  takes a different meaning that we will discuss in section 6. The proof of the SSFT was extended to stochastic dynamics in [2], [26]–[29]. Furthermore van Zon and Cohen proved that there is an important difference between the FTs for the injected power and those for the dissipated power [2, 29]. The SSFT has been proved also for other quantities such as the dissipation function [30] and the total entropy [31, 32]. Other theoretical papers studied the FT and the reader may find a review in [33, 34]. Experiments searching for FTs have been performed on dynamical systems [4, 5, 36], but interpretations are very difficult because a quantitative comparison with the theoretical prediction can be doubtful. Other experiments have been performed in stochastic systems described by a first-order Langevin equation: a Brownian particle in a moving optical trap [37] and an out-of-equilibrium electrical circuit [3] in which existing theoretical predictions [2, 29] were verified. Other experimental tests for FTs have been performed on driven two-level systems [19] and on colloids [18].

#### 3.2. FTs for Gaussian variables

Let us suppose that the variable  $X_{\tau}$  has a Gaussian distribution of mean  $\langle X_{\tau} \rangle$  and variance  $\sigma_{X_{\tau}}^2$ . It is easy to show that in order to satisfy the FTs, the variable  $X_{\tau}$  must have the following statistical property:

$$\sigma_{X_{\tau}}^2 = 2k_{\rm B}T\langle X_{\tau}\rangle.\tag{14}$$

This is an interesting relation because it imposes that the relative fluctuations of  $X_{\tau}$  are

$$\frac{\sigma_{X_{\tau}}}{X_{\tau}} = \sqrt{\frac{2k_{\rm B}T}{\langle X_{\tau} \rangle}}.$$
(15)

This means that the probability of having negative events reduces on increasing  $X_{\tau}$ ; specifically from equation (15) it follows that  $P(X_{\tau} < 0) = \operatorname{erfc}(\sqrt{\langle X_{\tau} \rangle / (2k_{\mathrm{B}}T)})$  where erfc is the complementary error function. It is now possible to estimate the length of the time interval  $t_{\mathrm{obs}}$  needed to observe at least one negative event, which is

$$t_{\rm obs} = \frac{\tau}{\operatorname{erfc}(\sqrt{\langle X_\tau \rangle / 2k_{\rm B}T})} \tag{16}$$

where we used the fact that all the values  $W_{\tau}$  computed on different intervals of length  $\tau$  are independent, which is certainly true if  $\tau$  is larger than the correlation time.

Let us consider the specific example of section 2.3, i.e.  $\langle W_{\tau} \rangle = 0.04n(k_{\rm B}T)$  at  $\omega_{\rm d}/(2\pi) = 67$  Hz,  $M_{\rm o} = 0.78$  pN m and  $\tau = 2\pi n/\omega_{\rm d}$ . The pdfs of  $W_{\tau}$  are Gaussian in this case (figure 6) and, as we will see in section 3.3, they satisfy the SSFT for large  $\tau$ . Therefore equation (15) holds for  $X_{\tau} = W_{\tau}$  and we may estimate  $t_{\rm obs}$  in the asymptotic limit  $\tau_{\alpha} \ll \tau$ . For example at n = 200, one obtains from the above mentioned experimental values  $\tau \simeq 3s \gg \tau_{\alpha}$  and  $\langle W_{\tau} \rangle = 8k_{\rm B}T$ . Inserting these experimental values in equation (15) one gets roughly a negative event over an observational time  $t_{\rm obs} \simeq 641$  s, which is already a rather long time for the distance between two events. For larger n and larger  $M_0$  this time becomes exponentially large. This explains the fact that millions of data are necessary in order to have a reliable measurement of SSFT.

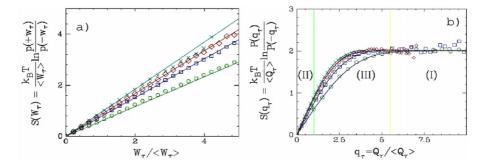
#### 3.3. FTs for $W_{\tau}$ and $Q_{\tau}$ measured in the harmonic oscillator

The questions that we ask are whether for finite time FTs are satisfied for either  $x_{\tau} = w_{\tau}$ or  $x_{\tau} = q_{\tau}$  and what the finite time corrections are. At a first time, we test the correction to the proportionality between the symmetry function  $S(x_{\tau})$  and  $x_{\tau}$ . In the region where the symmetry function is linear with  $x_{\tau}$ , we define the slope  $\Sigma_x(\tau)$ , i.e.  $S(x_{\tau}) = \Sigma_x(\tau)x_{\tau}$ . At a second time we measure finite time corrections to the value  $\Sigma_x(\tau) = 1$  which is the asymptotic value expected from FTs.

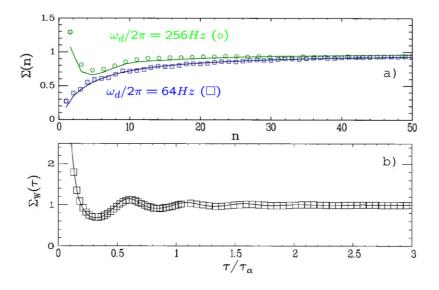
In this review paper we will focus on the SSFT applied to the experimental results of section 2.3 and to other examples. The TFT will be not discussed here and the interested readers may look at [17].

From the pdfs of  $w_{\tau}$  and  $q_{\tau}$  plotted in figures 7, we compute the symmetry functions defined in equation (10). The symmetry functions  $S(w_n)$  are plotted in figure 8(a) as a function of  $w_n$ . They are linear in  $w_n$ . The slope  $\Sigma_w(n)$  is not equal to 1 for all n but there is a correction at finite time (figure 9(a)). Nevertheless,  $\Sigma_w(n)$  tends to 1 for large n. Thus SSFT is satisfied for  $W_{\tau}$  and for a sinusoidal forcing. The convergence is very





**Figure 8.** Sinusoidal forcing. Symmetry functions for SSFT. (a) Symmetry functions  $S(w_{\tau})$  plotted as a function of  $w_{\tau}$  for various values of n: n = 7 (O), n = 15 ( $\Box$ ), n = 25 ( $\diamondsuit$ ) and n = 50 ( $\times$ ). For all n the dependence of  $S(w_{\tau})$  on  $w_{\tau}$  is linear, with slope  $\Sigma_w(\tau)$ . (b) Symmetry functions  $S(q_{\tau})$  plotted as a function of  $q_{\tau}$  for various values of n. The dependence of  $S(q_{\tau})$  on  $q_{\tau}$  is linear only for  $q_{\tau} < 1$ . Continuous lines are theoretical predictions.



**Figure 9.** Finite time corrections for the SSFT. (a) Sinusoidal forcing.  $\Sigma_w(\tau)$  as a function of n obtained from the slopes of the straight lines of figure 8(a) ( $\Box$ ). The circles correspond to another measurement performed at a different frequency. The finite time corrections depend on the driving frequency. The slopes  $\Sigma_q(\tau)$  measured for  $q_{\tau} < 1$  (figure 8(b)) have exactly the same values of  $S_w(\tau)$  as a function of n. (b) Linear forcing.  $\Sigma_w(\tau)$  measured as a function of  $\tau$  with the driving torque M has a linear dependence on time. The finite time corrections depend on the driving.

slow and we have to wait a large number of periods of forcing for the slope to be 1 (after 30 periods, the slope is still 0.9). This behavior is independent of the amplitude of the forcing  $M_{\rm o}$  and consequently of the mean value of the work  $\langle W_n \rangle$ , which, as explained in section 3.2, changes only the time needed to observe a negative event. The system satisfies the SSFT for all forcing frequencies  $\omega_{\rm d}$  but finite time corrections depend on  $\omega_{\rm d}$ , as can be seen in figure 9(a).

We now analyze the pdf of  $q_{\tau}$  (figure 7(b)) and we compute the symmetry functions  $S(q_n)$  of  $q_n$  plotted in figure 8(b) for different values of n. They are clearly very different from those of  $w_n$  plotted in figure 8(a). For  $S(q_n)$  three different regions appear:

- (I) For large fluctuations  $q_n$ ,  $S(q_n)$  equals 2. When  $\tau$  tends to infinity, this region spans from  $q_n = 3$  to infinity.
- (II) For small fluctuations  $q_n$ ,  $S(q_n)$  is a linear function of  $q_n$ . We then define  $\Sigma_q(n)$  as the slope of the function  $S(q_n)$ , i.e.  $S(q_n) = \Sigma_q(n) q_n$ . We have found by measurement [17] that  $\Sigma_q(n) = \Sigma_w(n)$  for all the values of n, i.e. finite time corrections are the same for heat and work. Thus  $\Sigma_q(n)$  tends to 1 when  $\tau$  is increased and SSFT holds in this region II which spans from  $q_n = 0$  up to  $q_n = 1$  for large  $\tau$ . This effect was discussed for the first time in [29, 2].

(III) A smooth connection between the two behaviors.

These regions define the fluctuation relation from the heat dissipated by the oscillator. The limit for large  $\tau$  of the symmetry function  $S(q_{\tau})$  is rather delicate and it has been discussed in [17].

The conclusions of this experimental analysis is that SSFT holds for work for any value of  $w_{\tau}$  whereas for heat it holds only for  $q_{\tau} < 1$ . The finite time correction to FTs, described by  $1-\Sigma$ , are not universal. They are the same for both  $w_{\tau}$  and  $q_{\tau}$  but they depend on the driving frequency as shown in figure 9(a). Furthermore they depend on the kind of driving force. In figure 9(b) we plot  $\Sigma_w(\tau)$  measured when the harmonic oscillator is driven out of equilibrium by a linear ramp<sup>1</sup>. The difference with respect to figure 9(a) is quite evident.

#### 3.4. Comparison with theory

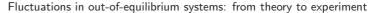
This experimental analysis allows a very precise comparison with theoretical predictions using the Langevin equation (equation (5)) and using two experimental observations: (a) the properties of heat bath are not modified by the driving and (b) the fluctuations of the  $W_{\tau}$  are Gaussian (see also [38], where it is shown that in Langevin dynamics  $W_{\tau}$  has a Gaussian distribution for any kind of deterministic driving force if the properties of the bath are not modified by the driving and the potential is harmonic). The observation in point (a) is extremely important because it is always assumed to be true in all the theoretical analysis. In [17] this point has been precisely checked. Using these experimental observations one can compute the pdf of  $q_{\tau}$  and the finite time corrections  $\Sigma(\tau)$  to the SSFT (see [17]). The continuous lines in figures 9, 8 and 6 are not fits but analytical predictions, with no adjustable parameters, derived from the Langevin dynamics of equation (5) (see [17] for more details).

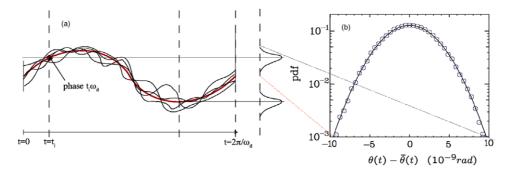
#### 3.5. The trajectory-dependent entropy

In previous sections we have studied the energy  $W_{\tau}$  injected into the system in the time  $\tau$  and the energy dissipated towards the heat bath  $Q_{\tau}$ . These two quantities and the internal energy are related by the first law of thermodynamics (equation (9)). Following the notation of [31], we define the entropy variation in the system during a time  $\tau$  as

$$\Delta s_{\mathrm{m},\tau} = \frac{1}{T} Q_{\tau}.$$
(17)

<sup>1</sup> The stationarity in the case of a ramp is discussed in [29, 17].





**Figure 10.** (a) Schematic diagram illustrating the method used to compute the trajectory-dependent entropy; (b) pdf of  $\theta(t)$  around the mean trajectory  $\langle \theta(t) \rangle$ . The continuous line is the equilibrium distribution.

For thermostated systems, entropy change in a medium behaves like the dissipated heat. The non-equilibrium Gibbs entropy is

$$\langle s(t) \rangle = -k_{\rm B} \int \mathrm{d}\vec{x} \, p(\vec{x}(t), t, \lambda_t) \ln p(\vec{x}(t), t, \lambda_t) \tag{18}$$

where  $\lambda_t$  denotes the set of control parameters at time t and  $p(\vec{x}(t), t, \lambda_t)$  is the probability density function for finding the particle at a position  $\vec{x}(t)$  at time t, for the state corresponding to  $\lambda_t$ . This expression allows the definition of a 'trajectory-dependent' entropy:

$$s(t) \equiv -k_{\rm B} \ln p(\vec{x}(t), t, \lambda_t). \tag{19}$$

The variation  $\Delta s_{\text{tot},\tau}$  of the total entropy  $s_{\text{tot}}$  during a time  $\tau$  is the sum of the entropy change in the system during  $\tau$  and the variation of the 'trajectory-dependent' entropy in a time  $\tau$ ,  $\Delta s_{\tau} \equiv s(t+\tau) - s(t)$ :

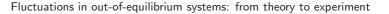
$$\Delta s_{\text{tot},\tau} \equiv s_{\text{tot}}(t+\tau) - s_{\text{tot}}(t) = \Delta s_{\text{m},\tau} + \Delta s_{\tau}.$$
(20)

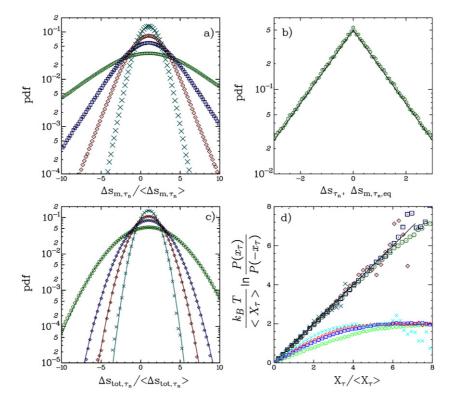
In this section, we study fluctuations of  $\Delta s_{\text{tot},\tau}$  computed using (17) and (19). We will show that  $\Delta s_{\text{tot},\tau}$  satisfies a SSFT for all  $\tau$ . In [32], the relevance of boundary terms like  $\Delta s_{\tau}$  has been pointed out for Markovian processes.

We investigate the data of the harmonic oscillator described in section 2.3. The probability to compute is the joint probability  $p(\theta(t_i + \tau_n), \dot{\theta}(t_i + \tau_n), \varphi)$ , where  $\varphi$  is the starting phase  $\varphi = t_i \omega_d$ . The system is linear, so  $\theta(t_i + \tau_n), \dot{\theta}(t_i + \tau_n)$  are independent; thus the joint probability can be factorized into a product. The expression for the trajectory-dependent entropy is

$$\Delta s_{\tau_n} = -k_{\rm B} \ln \left( \frac{p(\theta(t_i + \tau_n), \varphi) p(\dot{\theta}(t_i + \tau_n, \varphi))}{p(\theta(t_i), \varphi) p(\dot{\theta}(t_i, \varphi))} \right).$$
(21)

For computing correctly the trajectory-dependent entropy, we have to calculate the  $p(\theta(t_i), \varphi)$  and  $p(\dot{\theta}(t_i), \varphi)$  for each initial phase  $\varphi$  (see figure 10(a)). These distributions turn out to be independent of  $\varphi$  and they correspond to the equilibrium fluctuations of  $\theta$  and  $\dot{\theta}$  around the mean trajectory defined by  $\langle \theta(t) \rangle$  and  $\langle \dot{\theta}(t) \rangle$ . The distribution of  $\theta(t_i)$  is plotted in figure 10(b), where the continuous line corresponds to the

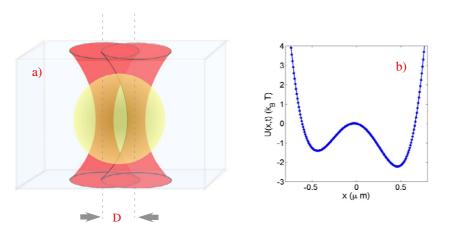




**Figure 11.** Torsion pendulum. (a) The pdfs of the normalized entropy variation  $\Delta s_{m,\tau_n}/\langle \Delta s_{m,\tau_n} \rangle$  integrated over *n* periods of forcing, with n = 7 (O), n = 15 (D), n = 25 ( $\diamond$ ) and n = 50 (×). (b) The pdfs of  $\Delta s_{\tau_n}$ ; the distribution is independent of *n* and here n = 7. The continuous line is the theoretical prediction for equilibrium entropy exchanged with thermal bath  $\Delta s_{m,\tau_n,eq}$ . (c) The pdfs of the normalized total entropy  $\Delta s_{\text{tot},\tau_n}/\langle \Delta s_{\text{tot},\tau_n} \rangle$ , with n = 7 (O), n = 15 (D), n = 25 ( $\diamond$ ) and n = 50 (×). (d) Symmetry functions for the normalized entropy variation in the system (small symbols in light colors and  $X_{\tau}$  standing for  $T\Delta s_{m,\tau_n} = Q_{\tau}$ ) and for the normalized total entropy (large symbols in dark colors and  $X_{\tau}$  standing for  $T\Delta s_{\text{tot},\tau_n}$ ) for the same values of *n*.

equilibrium distribution. Once the  $p(\theta(t_i, \varphi) \text{ and } p(\dot{\theta}(t_i), \varphi))$  are determined we compute the 'trajectory-dependent' entropy. As fluctuations of  $\theta$  and  $\dot{\theta}$  are independent of  $\varphi$  we can average  $\Delta s_{\tau_n}$  over  $\varphi$  which improves the statistical accuracy a lot. We stress that it is not equivalent to calculate first the pdfs over all values of  $\varphi$ —which would correspond here to the convolution of the pdf of the fluctuations with the pdf of a periodic signal—and then compute the trajectory-dependent entropy. The results are shown in figure 11.

In figure 11(a), we recall the main results for the dissipated heat  $Q_{\tau} = T\Delta s_{m,\tau_n}$ . Its average value  $\langle T\Delta s_{m,\tau_n} \rangle$  is linear in  $\tau_n$  and equal to the injected work. The pdfs of  $T\Delta s_{m,\tau_n}$  are not Gaussian and extreme events have an exponential distribution. The pdf of the 'trajectory-dependent' entropy is plotted in figure 11(b); it is exponential and independent of n. We superpose on it the pdf of the variation of internal energy divided by T at equilibrium: the two curves match perfectly within experimental errors, so the 'trajectory-dependent' entropy can be considered as the entropy exchanged with



**Figure 12.** (a) Drawing of the polystyrene particle trapped by two laser beams whose axial distance is about the radius of the bead. (b) Potential felt by the bead trapped by the two laser beams. The barrier height between the two wells is about  $2k_{\rm B}T$ .

the thermostat if the system is at equilibrium. The average value of  $\Delta s_{\tau_n}$  is zero, so the average value of the total entropy is equal to the average of injected power divided by T. In figure 11(c), we plot the pdfs of the normalized total entropy for four typical values of the integration time. We find that the pdfs are Gaussian for any time.

The symmetry functions (equation (10)) of the dissipated heat  $S(T\Delta s_{m,\tau_n} = Q_{\tau})$  and the total entropy  $S(T\Delta s_{tot,\tau_n})$  are plotted in figure 11(d). As we have already seen in figure 8,  $S(Q_{\tau})$  is a non-linear function of  $Q_{\tau} = T\Delta s_{m,\tau}$ . The linear behavior, with a slope that tends to 1 for large time, is observed only for  $\Delta s_{m,\tau_n} < \langle \Delta s_{m,\tau_n} \rangle < 1$ . For the normalized total entropy, the symmetry functions are linear with  $\Delta s_{tot,\tau_n}$  for all values of  $\Delta s_{tot,\tau_n}$  and the slope is equal to 1 for all values of  $\tau_n$ . Note that this is not exactly the case for the first values of  $\tau_n$  because these are the times over which the statistical errors are the largest and the error in the slope is large.

For the harmonic oscillator we have obtained that the 'trajectory-dependent' entropy can be considered as the entropy variation in the system in a time  $\tau$  that one would have if the system was at equilibrium. Therefore the total entropy is the additional entropy due to the presence of the external forcing: this is the part of entropy which is created by the non-equilibrium stationary process. The total entropy (or excess entropy) satisfies the fluctuation theorem for all times and for all kinds of stationary external torque [31, 32]. More details on this problem can be found in [39].

#### 4. The non-linear case: stochastic resonance

The harmonic oscillator cannot be driven to a non-linear regime without forcing it to such a high level where thermal fluctuations become negligible. Thus in order to study the non-linear effects we change the experiment and we measure the fluctuations of a Brownian particle trapped in a non-linear potential produced by two laser beams, as shown in figure 12. It is very well known that a particle of small radius  $R \simeq 2 \ \mu m$  is trapped at the focus of a strongly focused laser beam, which produces a harmonic potential

for the particle, whose Brownian motion is confined inside this potential well. When two laser beams are focused at a distance  $D \simeq R$ , as shown in figure 12(a) the particle has two equilibrium positions, i.e. the foci of the two beams. Thermal fluctuations may force the particle to move from one to the other. The particle feels an equilibrium potential  $U_0(x) = ax^4 - bx^2 - dx$ , shown in figure 12(b), where a, b and d are determined by the laser intensity and by the distance apart of the two focal points. This potential has been computed from the measured equilibrium distribution of the particle  $P(x) \propto \exp(U_0(x))$ . The right–left asymmetry of the potential (figure 12(b)) is induced by small unavoidable asymmetries, induced by the optics focusing the two laser beams. In our experiment the distance between the two spots is 1.45  $\mu$ m, which produces a trap whose minima are at  $x_{\rm min} = \pm 0.45 \ \mu {\rm m}$ . The total intensity of the laser is 29 mW on the focal plane which corresponds to an inter-well barrier energy  $\delta U_{\rm o} = 1.8k_{\rm B}T$ ,  $ax_{\rm min}^4 = 1.8k_{\rm B}T$ ,  $bx_{\rm min}^2 = 3.6k_{\rm B}T$ and  $d|x_{\min}| = 0.44k_{\rm B}T$  (see [40] for more experimental details). The rate at which the particle jumps from one potential well to the other is determined by the Kramers rate  $r_k = (1/\tau_0) \exp(-\delta U_0/k_{\rm B}T)$  where  $\tau_0$  is a characteristic time. In our experiment,  $r_k \simeq 0.3$  Hz at 300 K.

To drive the system out of equilibrium we periodically modulate the intensity of the two beams at low frequency. Thus the potential felt by the bead is the following profile:

$$U(x,t) = U_0(x) + U_p(x,t) = U_0 + cx\sin(2\pi ft),$$
(22)

with  $c|x_{\min}| = 0.81 \ k_{\rm B}T$ . The amplitude of the time-dependent perturbation is acquired synchronously with the bead trajectory<sup>2</sup>.

An example of the measured potential for t = (1/4f) and (3/4f) is shown in figure 13(a). This figure is obtained by measuring the probability distribution function P(x,t) of x for fixed values of  $c \sin(2\pi ft)$ ; it follows that  $U(x,t) = -\ln(P(x,t))$ .

The x position of the particle can be described by a Langevin equation:

$$\gamma \dot{x} = -\frac{\partial U(x,t)}{\partial x} + \eta, \qquad (23)$$

with  $\gamma = 1.61 \times 10^{-8}$  N s m<sup>-1</sup> the friction coefficient and  $\eta$  the thermal noise delta correlated in time. When  $c \neq 0$  the particle can experience a stochastic resonance [41], when the forcing frequency is close to the Kramers rate. An example of the sinusoidal force with the corresponding position are shown in figure 13(b). Since the synchronization is not perfect, sometimes the particle receives energy from the perturbation, and sometimes the bead moves against the perturbation leading to a negative work on the system. Two kinds of work can be defined in this experiment [40]:

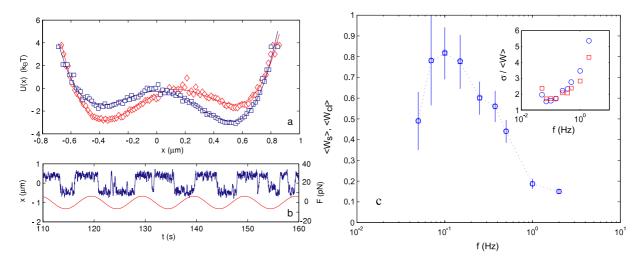
$$W_{s,n}(t) = \int_{t}^{t+t_f} \mathrm{d}t \frac{\partial U(x,t)}{\partial t}$$
(24)

$$W_{cl,n}(t) = -\int_{t}^{t+t_{f}} \mathrm{d}t \, \dot{x} \frac{\partial U_{p}(x,t)}{\partial x}$$
(25)

where in this case  $t_f = n/f$  is a multiple of the forcing period. The work  $W_{s,n}$  is the stochastic work (used in Jarzynski and Crooks relations [47, 49, 13]) and  $W_{cl,n}$  is the

<sup>2</sup> The parameters given here are average parameters since the coefficients a, b and c, obtained from fitted steady distributions at given phases, vary with the phase ( $\delta a/a \approx 10\%$ ,  $\delta b/b \approx \delta c/c \approx 5\%$ ).



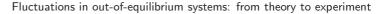


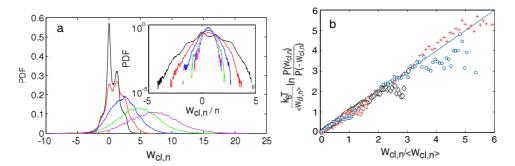
**Figure 13.** (a) The perturbed potential at t = (1/4f) and t = (3/4f). (b) Example of a trajectory of the glass bead and the corresponding perturbation at f = 0.1 Hz. (c) Mean injected energy in the system over a single period as a function of the driving frequency.  $\langle W_s \rangle \Box$  and  $\langle W_{cl} \rangle \odot$  coincide as their mean values are equal within experimental errors. The error bars are computed from the standard deviation of the mean over different runs. Inset: standard deviations of work distributions over a single period normalized by the average work as a function of the frequency (same symbols).

classical work that will be discussed in this paper. The results for  $W_{s,n}$  are quite similar but there are subtle differences discussed in [40].

We first measure the average work received over one period for different frequencies  $(t_f = 1/f$  in equation (25)). Each trajectory is here recorded during 3200 s in different consecutive runs, which corresponds to 160 to 6400 forcing periods for the range of frequencies explored. In order to increase the statistics we consider  $10^5$  different values of  $t_0$ . Figure 13(c) shows the evolution of the mean work per period for both definitions of the work. First, the input average work decreases to zero when the frequency tends to zero. Indeed, the bead hops randomly several times between the two wells during the period. Second, in the limit of high frequencies, the particle has no time to jump onto the other side of the trap but rather stays in the same well during the period; thus the input energy is again decreasing with increasing frequency. In the intermediate regime, the particle can almost synchronize with the periodic force and follows the evolution of the potential. The maximum of injected work is found around the frequency  $f \approx 0.1$  Hz, which is comparable with half of the Kramers rate of the fixed potential  $r_{\rm K} = 0.3$  Hz. This maximum of transferred energy shows that the stochastic resonance for a Brownian particle is a bona fide resonance, as was previously shown in experiments using resident time distributions [42, 43] or directly in simulations [44, 45]. In the inset of figure 13, we plot the normalized standard deviation of work distributions  $(\sigma/\langle W \rangle)$  as a function of the forcing frequency. The curves present a minimum at the same frequency of 0.1 Hz, in agreement again with the resonance phenomena.

In order to study the FT for stochastic resonance we choose for the external driving a frequency f = 0.25 Hz, which ensures a good statistic, by allowing the observation of the





**Figure 14.** (a) Distribution of classical work  $W_{cl}$  for different numbers of periods n = 1, 2, 4, 8 and 12 (f = 0.25 Hz). Inset: the same data in a linear-log form. (b) Normalized symmetry function as a function of the normalized work for n = 1 (+), 2 ( $\bigcirc$ ), 4 ( $\diamondsuit$ ), 8 ( $\bigtriangleup$ ), 12 ( $\square$ ).

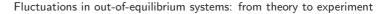
system over a sufficient number of periods. We compute the works and the dissipation using  $1.5 \times 10^6$  different values of t in a time series which spans about 7500 periods of the driving.

We consider the pdf  $P(W_{cl})$  which is plotted in (figure 14(a)). Notice that for small n the distributions are double peaked and very complex. They tend to a Gaussian for large n (inset of figure 14(a)). In figure 14(b), we plot the normalized symmetry function of  $W_{cl,n}$ . We can see that the curves are close to the line of slope 1. For high values of work, the dispersion of the data increases due to the lack of events. The slope tends toward 1 as expected by the SSFT. It is remarkable that straight lines are obtained even for n close to 1, where the distribution presents a very complex and unusual shape (figure 14(a)). We do not discuss here the case of  $W_{s,n}$  as the behavior is quite similar to that of  $W_{cl,n}$  [40]. The very fast convergence to the asymptotic value of the SSFT is quite striking in this example. The measurements are in full agreement with a realistic model based on the Fokker–Planck equations where the measured value of U(x, t) has been inserted [46]. This example shows the application of the FT in a non-linear case where the distributions are strongly non-Gaussian.

#### 5. Applications of fluctuation theorems

The fluctuations theorems have several important consequences such as the Jarzynski and Crooks equalities [47]–[49], which are useful for computing the free energy difference between two equilibrium states using any kind of transformation [13, 15, 51, 50]. The Hatano–Sasa [22] relations and the recently derived fluctuation-dissipation theorems [52, 53] are related to FTs and are useful for computing the response of a NESS using the steady state fluctuations of the NESS. As we have seen, the FT allows the calculation of tiny amounts of heat, which can be useful in many applications in ageing systems [54, 55] and biological systems.

The FTs for Langevin systems can be used to measure an unknown averaged power. This idea was discussed first in the context of electrical circuits [3] and in [58] it was applied for the first time to the measuring of the torque of a molecular motor. We discuss the method in some detail in section 5.1.



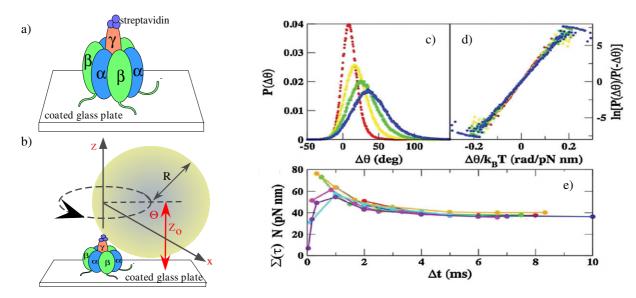


Figure 15. Molecular motor. (a) A schematic diagram of an F<sub>1</sub>-ATPase molecular motor composed of a rotor  $\gamma$  (radius ~1 nm) which rotates inside a stator of radius ~5 nm formed by three  $\alpha\beta$  subcomplexes. Sequential chemical reactions between the stator and the rotor produce the motion. The  $\alpha\beta$  subcomplexes are attached to a suitably coated glass plate. Streptavidin is used to attach to  $\gamma$  either actin filaments [59] or streptavidin-coated beads [58]. (b) In order to follow the rotation of the rotor with a standard microscope a streptavidin-coated bead of radius ~0.5  $\mu$ m is glued to a  $\gamma$  subunit (drawing not to scale). Parts (c)– (e) (taken from [58]) illustrate the results of a measurement. (c) The pdfs for several values  $\tau$  of  $\Delta\theta_{\tau}$ . (d) The symmetry function extracted from the pdfs of (c). (e) Slopes  $N\Sigma(\tau)$  of the symmetry function as a function of  $\tau$ . The different colors pertain to different experimental conditions. Notice the convergence to an unique value of N for large  $\tau$ . Strictly speaking, in this figure the function  $\Sigma(\tau)$ takes into account also the fact that equation (26) is not necessarily valid for short times (see the text).

#### 5.1. Measuring the power of a molecular motor

A molecular rotary motor, like any kind of motor, is constituted by a stator and a rotor. The movement of the rotor is provoked by chemical reactions occurring sequentially between the rotor and stator. A typical example of bio-motor is the bacterial flagellum. However it has been shown [59] that a single molecule of  $F_1$ -ATPase may act as a motor composed by a  $\gamma$  subunit (radius: ~1 nm) which rotates inside a barrel of radius ~5 nm formed by three  $\alpha\beta$  subcomplexes (see figure 15(a) for a schematic diagram and [58,59] for more details). In experiments, the  $\alpha\beta$  subunit is stuck on a suitably activated glass plate as shown in figure 15(a). The measuring of the torque of this motor is important for establishing its efficiency as a function of the concentration of the chemicals contained in the liquid surrounding it. The typical size of this molecular motor is several nanometers and the moving unit is too small to be observed with an optical microscope. Therefore to measure the torque of the F<sub>1</sub>-ATPase motor a streptavidin-coated bead of radius 0.5  $\mu$ m is glued on the subunit  $\gamma$ , and the motion of this bead is followed by a standard microscope

as sketched in figure 15(b). The motion of the bead occurs on a torus and  $\theta$  is the coordinate of the motion along the torus. The time evolution of  $\theta$  can be described using a Langevin equation:

$$\Gamma \theta = N + \xi, \tag{26}$$

where N is the torque of the motor,  $\Gamma$  the viscous dissipation of the fluid surrounding the bead and  $\xi$  the thermal noise. In order to estimate N, it is in principle enough to measure the mean angular velocity  $\langle \dot{\theta} \rangle$  of the bead and obviously from equation (26) one gets

 $N = \Gamma \langle \dot{\theta} \rangle. \tag{27}$ 

However the procedure is not so simple because in order to estimate N one has to know the exact value of  $\Gamma$ , which is a function of the viscosity of the fluid, the radius L of the bead path, the radius R of the bead, the shape of the bead and the distance  $Z_o$  of the bead from the surface of the glass plate where the experiment is performed (see figure 15(b)). The variable  $Z_o$  is certainly the most difficult to determine experimentally. Therefore using equation (27) the error on N can be really very large. There is however another method proposed in [3] for electric circuits and first used for measurements on bio-motors in [58]. This method is based on work fluctuations and is much more precise. To apply it, we suppose that N is constant. This is a quite reasonable hypothesis for certain regimes of  $F_1$ -ATPase motors. We compute the work  $W_{\tau}$  performed by the motor in a time  $\tau$ :

$$W_{\tau}(t) = \int_{t}^{t+\tau} N\dot{\theta}(t) \,\mathrm{d}t = N\Delta\theta_{\tau} \tag{28}$$

where  $\Delta \theta_{\tau} = \theta(t + \tau) - \theta(t)$  and we have used the fact that N is constant. In section 5 we have seen that Langevin systems satisfy the SSFT, which we now apply to  $W_{\tau}$ . Using equation (28) and the constancy of N, the SSFT for the molecular motor reads

$$\ln\left(\frac{P(\Delta\theta_{\tau})}{P(-\Delta\theta_{\tau})}\right) = \Sigma(\tau)N\frac{\Delta\theta_{\tau}}{k_{\rm B}T} \qquad \text{with } \Sigma(\tau) \to 1 \quad \text{for } \tau \to \infty.$$
(29)

This equation is quite interesting because the value of N can be determined only by measuring the fluctuations of  $\Delta \theta_{\tau}$ . Indeed plotting  $\ln(P(\Delta \theta_{\tau})/P(-\Delta \theta_{\tau}))$  as a function of  $\Delta \theta_{\tau}/(k_{\rm B}T)$  we notice that the slope of the straight lines is  $\Sigma(\tau)N$ . Therefore studying the asymptotic value of this slope for large  $\tau$  one can determine N. It is interesting to note that in this case the knowledge of  $\Gamma$  is not needed. This technique has been recently applied to molecular motor in [58] and their main results are plotted in figures 15(c)– (e). The relevant parameter  $\Sigma(\tau)N$ , extracted from the pdf of  $\Delta \theta_{\tau}$  (figure 15(c)) and the symmetry function (figure 15(d)), is plotted in figure 15(e).<sup>3</sup> We see a clear convergence to a unique value and one gets a very precise estimation of the torque of the molecular motor independently of the size and shape of the bead glued to the  $\gamma$  unit of the F<sub>1</sub>-ATPase motor. This is a very specific and interesting example of the possible applications of the FT.

<sup>&</sup>lt;sup>3</sup> It has to be stressed that in this specific case  $\Sigma(\tau)$  takes into account the fact that for short time, equation (26) is not a good approximation for the dynamics of the motor and equations (28) and (29) apply only for long time.

#### 6. The chaotic systems

In previous sections we have studied the probability of instantaneous negative entropy production rates within the context of the FTs for stochastic systems, where the fluctuations are produced by the coupling with a thermal bath. In section 3.2 we have seen that when the energy injected into the system is larger than  $100k_{\rm B}T$  the probability of these negative events is very small and the time needed to observe them becomes extremely long. In other words the role of thermal fluctuations becomes negligible.

However in the introduction we have shown that instantaneous negative entropy production rates can be observed in chaotic systems such as, for example, turbulent and granular media, where the fluctuations are produced by the non-linear interactions of many degrees of freedom. We have also pointed out that for chaotic systems the amount of injected energy is orders of magnitude larger than  $k_{\rm B}T$  and of course thermal fluctuations do not play any role in the fluctuating dynamics. The question that we want to analyze in this section is that of whether we can apply in these systems the FTs defined in section 3 for stochastic systems, equations (10)–(13). For a dissipative chaotic system one could imagine replacing  $k_{\rm B}T$ , in equations (10)–(13), with a characteristic energy  $E_{\rm c}$  which takes into account the relevant energy scales of the system fluctuations. However the definition of this relevant energy scale can be in general difficult and even impossible, because it may depend on the observable and on the kind of forcing. Thus the approach of introducing an  $E_{\rm c}$  is not very useful for comparing the experimental results with the proof given for dynamical systems [25]. Indeed in this case the theorem considers a quantity

$$y_{\tau} = \frac{\sigma_{\tau}}{\langle \sigma_{\tau} \rangle} = \frac{\int_{t}^{t+\tau} \sigma(t) \,\mathrm{d}t}{\langle \sigma_{\tau} \rangle} \tag{30}$$

where  $\sigma(t)$  is the instantaneous phase space contraction rate,  $\sigma_{\tau}$  the integral of  $\sigma$  for a time  $\tau$  and  $\langle \sigma_{\tau} \rangle$  the mean of  $\sigma_{\tau}$ . Three hypotheses have been advanced for the dynamical system which must be: (a) dissipative, (b) time reversible, (c) Anosov<sup>4</sup>.

$$\frac{1}{\tau} \ln \frac{P(y_{\tau})}{P(-y_{\tau})} = \langle \sigma \rangle y_{\tau} + O(1/\tau) \qquad \text{for } \tau \to \infty$$
(31)

where  $\langle \sigma \rangle$  is the mean phase space contraction rate, which has the dimension of 1/t. In this equation the relevant variable is the phase space contraction rate which has been identified as the entropy production rate [25]. The phase space contraction rate is a global variable of the system but an extension of the theorem for local variables has been given in [56, 57]. One reason for developing the local FT is that global fluctuations are usually not observable in macroscopic systems, and as a consequence it is important to understand whether a local measurement is representative of the dynamics. Equation (31) has been tested in several numerical simulations (see for example [33, 34] for a review); here we want to focus on experiments.

#### 6.1. Experimental testing

The testing of equation (31) in experiments is extremely useful for analyzing several important questions. The first one is whether equation (31) may have a more general

 $<sup>^{4}</sup>$  For a precise definition, see [35, 34]. Roughly speaking this property ensures that the system is chaotic and that on the attractor there are no regions of finite volume that do not contain points.

validity independently of the restrictive hypotheses used to prove it. Indeed hypothesis (b) is never satisfied in real systems and hypothesis (c) does not necessarily apply to all of them. Thus in general we do not even know whether equation (31) can be applied in the experimental system under study. The second question concerns the choice of the observable. Indeed the direct measuring of the phase space contraction rate is not possible and one has to rely upon the measuring of another observable, usually the energy  $W_{\tau}$  injected into the system by the external forces in a time  $\tau$ . In other words one is making the important hypothesis that  $y_{\tau}$ , defined in equation (30) in terms of  $\sigma(t)$ , is equivalent to  $x_{\tau} = W_{\tau}/\langle W_{\tau} \rangle$ . This hypothesis, which is not necessarily valid, provides the second question that one would like to address in experiments. The third question is related to the estimation of the prefactor  $\langle \sigma \rangle$  in the right-hand side of equation (31). This prefactor, which is a function of the Lyapunov exponents, is very difficult to estimate in an experimental system. Finally the last question concerns the relevance of a local observable for characterizing the dynamics of the system.

There are not many experiments where these questions have been analyzed in detail. In several experiments [4, 36] only the linearity in  $x_{\tau}$  of the symmetry function  $\rho(x_{\tau}) = (1/\tau) \ln(P(x_{\tau})/P(-x_{\tau}))$  has been checked, which, for the reasons discussed in the previous paragraph, is only a partial test. For example three experiments have tried to give an answer to the question regarding the prefactor. Two of these experiments are performed on granular media [5, 6] and the third on mechanical wave turbulence in a metallic plate [64]. We will not describe the experiments in detail here but we will comment the main results.

6.1.1. Granular media. The two experiments of [5,6] are on diluted granular media strongly shaken by a vibrator, but the measured quantities are not the same. In [5] the authors measure the fluctuations of the energy flux in a subvolume of the system. In [6], however, the work done by an external force on a ratchet inside the granular media is measured. Both studies find that although the system is not thermal, the stochastic version of the SSFT (equation (10)) holds, provided that  $k_{\rm B}T$  is replaced by a characteristic energy  $E_{\rm c}$ :

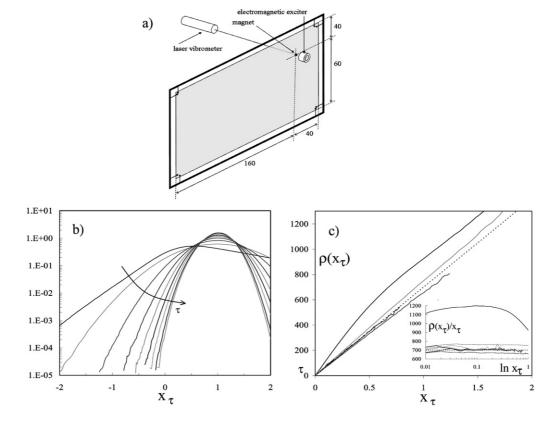
$$\ln \frac{P(x_{\tau})}{P(-x_{\tau})} = \frac{\langle X_{\tau} \rangle}{E_{c}} x_{\tau} + O(1/\tau)$$
(32)

with  $x_{\tau} = X_{\tau}/\langle X_{\tau} \rangle$ , where  $X_{\tau}$  is the integral of the energy flux in [5] and the work  $W_{\tau}$  performed by an external force on a ratchet in [6]. It must be pointed out that in both experiments of [5, 6] the energy  $E_c$  has been measured independently. In [5] it is found that  $E_c$  is about five times larger than the kinetic energy  $K_E$  of the shaken granular medium for all the values of the control parameters used in the experiment. The fact that  $K_E$  and  $E_c$  have the same dependence on the control parameter has been interpreted considering that the vibrator injects into the system the amount of energy lost in the collisions, but once excited in a NESS the granular medium behaves like a thermal bath for the measured observable. However this interpretation is not necessarily correct. Indeed for the experiment of [5] it as been shown in a numerical simulation [60] that for the quantity measured in this experiment the FT does not apply for the large deviations  $x_{\tau}$ because the symmetry function  $S(x_{\tau})$  becomes non-linear for large  $x_{\tau}$ . This discrepancy between theory and experiment is obviously arising from the fact that experimentally the very large deviation are difficult to measure; thus the non-linear part of  $S(x_{\tau})$  cannot be observed. However the experiment of [5] is certainly interesting because it is the first where the question of the prefactor has been analyzed experimentally. In contrast, for the experiment of [6] it is observed that  $E_{\rm c} = K_E(1+\alpha)/2$  where  $\alpha$  is the restitution coefficient of the grains. It is interesting to notice that  $K_E(1+\alpha)/2$  is the temperature of an intruder inside a diluted granular gas, as has been found in theoretical models [61]-[63]. These two examples of comparison between numerical and experimental results show the difficulty of interpreting the experimental results on the FT and the importance of verifying them in a precise theoretical framework. Where does the difference between the experiments of [5] and 6 come from? The answer can be found in the fact that in the two experiments two different quantities are measured. Indeed in section 3 we have seen that, even for stochastic systems, the fluctuations of  $W_{\tau}$ ,  $Q_{\tau}$  and  $\Delta s_{\text{tot},\tau_n}$  behave differently within the context of the FT. This can be more complex for granular media and it will be useful to gain more insight into this point. Furthermore even in cases where a description in terms of  $E_{\rm c}$  applies, comparing equation (32) with equation (31) a question that arises naturally is that of whether  $\langle W_{\tau} \rangle / E_{\rm c}$  is a good estimation of  $\langle \sigma \rangle \tau$ . This is an important question which will be interesting to analyze in the future.

6.1.2. Mechanical waves. In the experiment on mechanical waves [64], a metallic plate is set into a chaotic state of wave turbulence by a periodic local forcing at 75 Hz (see figure 16(a)). The chaotic dynamics is produced by the non-linear interaction of the oscillatory modes of the plate. The authors measure the local force and displacement (see figure 16(a)) and compute the work  $W_{\tau}$  done in a time  $\tau$  by the external force, which excites the vibrations of the plate. They find that the pdfs of  $x_{\tau} = W_{\tau}/\langle W_{\tau} \rangle$  are strongly non-Gaussian (see figure 16(b)). From these pdfs they compute the symmetry function  $\rho(x_{\tau})$  which is plotted (see figure 16(c)) as a function of  $x_{\tau}$ . We see that in spite of the fact that the pdfs are not Gaussian, the function  $\rho(x_{\tau})$  (figure 16(c)) converges to a unique straight line for large  $\tau$  as predicted by FT. From equation (31) the slope of this straight line is  $\langle \sigma \rangle$ , which the authors can estimate independently by measuring the relaxation time of the vibrational modes. They find that the values estimated with the two methods (using the FT and the relaxation time) are very close and within experimental errors. This result is quite interesting and it is probably the only experiment where a direct test of equation (31) has been made. Certainly the errors of this comparison are very large, but these kinds of tests are useful for understanding in some detail the applications of FTs to chaotic systems.

#### 7. Summary and concluding remarks

In this paper we have reviewed several experimental results on the fluctuations of injected and dissipated power in out-of-equilibrium systems. We considered the two cases when the fluctuations are produced by the coupling with the heat bath (stochastic systems) and when they are produced by the non-linear interactions of many degrees of freedom (chaotic systems). We have seen that in both cases we observe that the external forces may produce negative work because of fluctuations. The probability of these negative events has been analyzed in the framework of the fluctuation theorem.



Fluctuations in out-of-equilibrium systems: from theory to experiment

**Figure 16.** Mechanical waves in a metallic plate (from [64]). (a) A schematic diagram of the experiment. A steel plate is suspended to the frame. Dimensions are in centimeters. The electromagnetic exciter produces local forcing of the plate vibrations. A laser vibrometer measures the normal velocity at the excitation point. ((b), (c)) Results for the periodic forcing. (b) The pdfs of the injected power for the time durations  $\tau$  for  $\tau = 3.5$ , 6.5, 13.5, 20, 26.5, 33.5, 40, 47.5, 52 ms. (c) Functions  $\rho(x_{\tau}) = (1/\tau) \ln(P(x_{\tau})/P(-x_{\tau}))$  obtained from the pdfs of (b). Inset: the compensated value  $\rho(x_{\tau})/x_{\tau}$  in a semi-log plot. In (c) the dashed line corresponds to a linear law of slope  $\langle \sigma \rangle = 700$  Hz.

We have mainly discussed the stochastic systems described by using Langevin equations, both with harmonic and anharmonic potentials. We have seen that injected and dissipated powers present different behaviors. FTs are valid for any value of  $W_{\tau}$  whereas they can be applied only for  $Q_{\tau} < \langle Q_{\tau} \rangle$  in the case of the heat. We have also seen that the finite time corrections to the SSFT depend on the driving and on the properties of the system. We have introduced the total entropy, which takes into account only the entropy produced by the external forces neglecting the equilibrium fluctuations. For the total entropy, FTs are valid for all the times. We discussed the applications of FTs in extracting important physical properties of a stochastic system. Thus one may conclude that for Markovian systems driven by a deterministic force the application of the FT does not present any major problems and it can be safely applied. The case of random driving has been recently discussed and several problems may arise when the variance of the driving becomes larger than the fluctuations induced by the thermal bath. We have not discussed this problem, but an analysis of this specific case can be found in [8, 65, 66].

Finally we discussed the applications of the FT to chaotic systems. The experimental testing is in this case very important and useful because many questions can be asked as regards the system under study, which does not necessarily verify all the theoretical hypotheses. One has to say that in the case of non-Gaussian statistics even the linearity of the symmetry function can be an interesting result. However we pointed out that, for a real comparison with theory, the difficulty is in estimating the prefactor of equation (31) by using an independent measurement. Only a few experiments have addressed this point in some detail, but many problems remain open and it seems to be difficult to find a general behavior for chaotic systems as for the case of stochastic ones.

#### Acknowledgments

Several results reviewed in this paper were obtained in collaboration with L Bellon, R Gomez-Solano, N Garnier and P Jop. We acknowledge useful discussions and collaboration during the past ten years with D Andrieux, E Cohen, R Chetrite, G Gallavotti, P Gaspard, K Gawedzky and A Imparato. We thank A Puglisi and E Trizac for useful comments on the FT for granular media. We are grateful to H Noji, K Hayashi and A Boudauoud for allowing us to reproduce the figures from their articles in the present one.

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